

The Universal Vector Space and Static Equilibrium Techniques

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Abstract

In this paper, the author uses well know Structural Engineering theorems to solve for physical constants for the universe. The universe is in static equilibrium. Thus, these rules of statics apply to the universe. We show how the speed of light is derived to be $c=2.997$. In keeping with the theme of Astro-Theology, the physical universe are inseparable.

Keywords: Vector space; Force polygon; Static equilibrium; Speed of light

Introduction

Yet another way to analyze the universe is through a vector space, or a force polygon with 8 joints, 2 reactions, and 18 members. This system will be considered below (Figure 1).

The Universe can be modelled as a vector space, or alternatively named a force polygon in structural mechanics. Since there are 10 unknowns in AT math, there are 8 joints in this vector space plus two reactions acting at joint 3, Energy and time. There are 18 members connecting every nod to every other node. A member from joint 1 to joint 3 and joint 3 to joint 7 lines up perfectly, which indicates a statically determinate system. These two members will be analyzed below as a structural simply supported beam, what I term, the "universal beam." The load on these beams is q , and is the resultant of the Energy-time vectors. All the members of this vector space make up the physical constants of the universe (Figure 2).

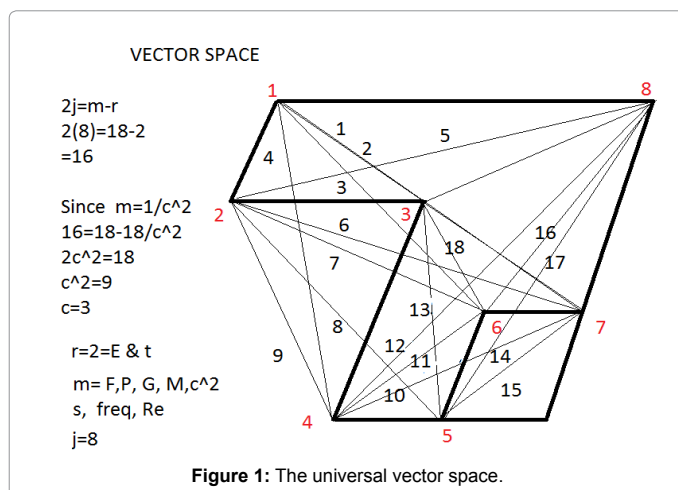
The definition of the numeral one is where the fraction meets the multiple. Every other number is either a multiple or a fraction of one. One, is therefore the tipping point. In algebraic terms, the fraction and the multiple meet at the golden mean. Therefore, $t^2-t-1=0$ is where Energy and time meet -at joint 3 shown in Figure 2.

LOAD:

$$q=f(E,t)$$

$$q=1+\sin 60, \text{ refer to Figure 3.}$$

$$q=t^2-t-1=E$$



$$1+\sin 60 = t^2-t-1=0$$

Roots,

$$t=2.2652, 1.2652=\rho$$

$$t=\rho \text{ when } E=q$$

ENERGY: refer to Figure 4.

$$E=Mc^2$$

$$dE/dt=dM/dt \ c^2$$

$$1=1/t=(2)(2)(3)$$

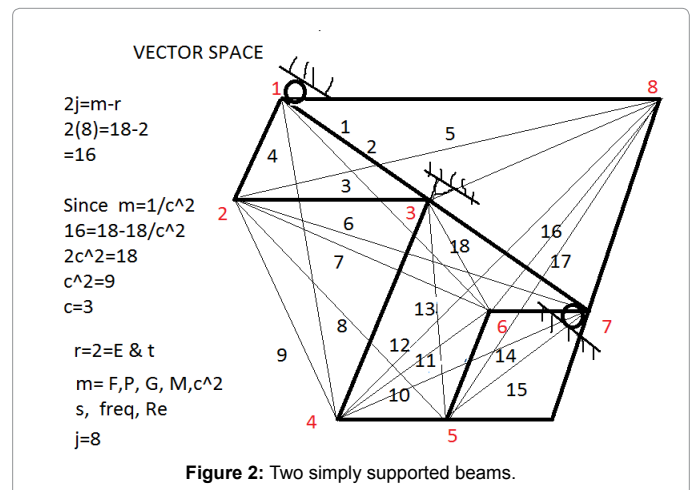
$$1/t=12$$

$$t=1/12$$

Cross Product

$$s=|E||t| \sin t$$

$$4/3=E \times (1/12)(\sin 60 \text{ deg.})$$

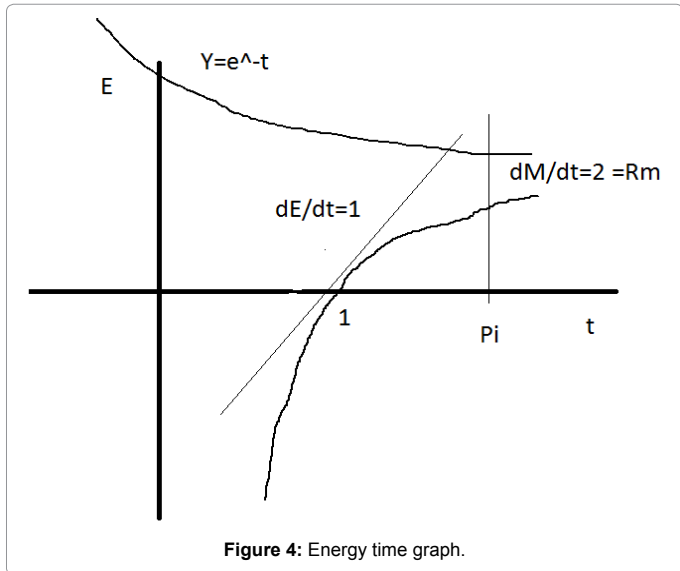
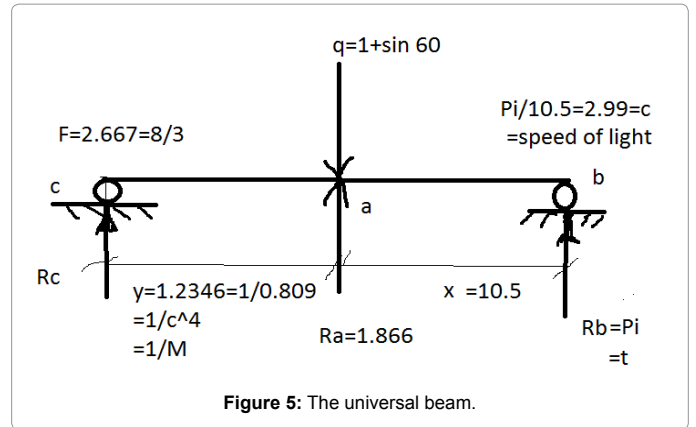
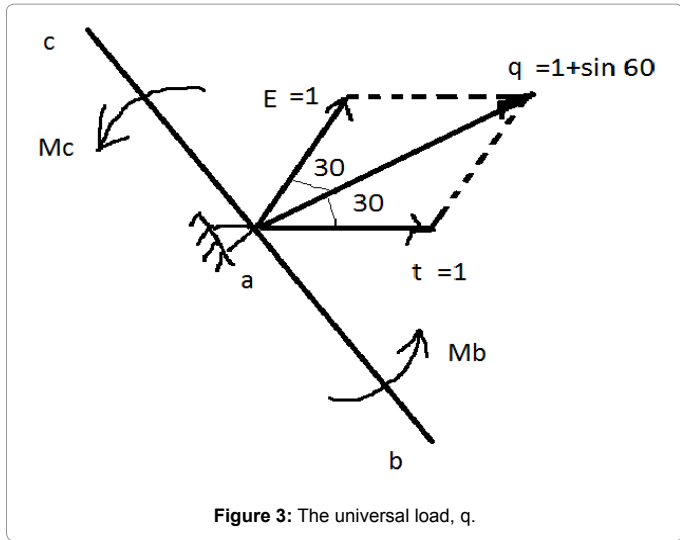


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When E=0

$$[(y^y)^y - y]/c = 0$$

$$(y^y)^y - y = 0$$

$$(dE/dt)^E - E = 0$$

Clairnaut

$$d^2E/dt^2 - E = 0$$

So,

$$(dE/dt)^E = d^2E/dt^2$$

$$(dy/dx)^y = d^2x^2/dt^2$$

$$y[\ln(dy/dx)] = \ln(d^2y^2/dt^2)$$

$$y(\ln 1) = y^c$$

$$y(0) = \ln y^c$$

$$0 = e^{\ln y^c}$$

$$0 = y^c$$

$$a = 0$$

$$F = Ma$$

$$F = 0$$

$$\Sigma F_x = 0 \text{ Static Equilibrium}$$

$$\Sigma F_x = 0$$

$$q \cos 60^\circ - F \cos 60^\circ = 1.866 (1/2) - (2.667 (1/2))$$

$$= -0.4005$$

$$= t$$

$$40\% \text{ of a cycle} = 1 \text{ rad.}$$

$$1/250 = 1/T = 1/[1/t] \text{ (Figure 6).}$$

$$f = My/I$$

$$0.0809 = M (\text{Pi}) / (1/Y)$$

$$M = 52$$

$$\text{Element} = \text{Tellurian at fulcrim}$$

$$\rho = 126.7$$

$$E = 1.0695 \sim 1.07 = \text{Mass H}^+$$

Add an electron:

$$1.07 + 0.511 = 1.581 \sim \sin 1$$

Energy & Graivitational Acceleration

$$E = W \times t$$

$$= F \times dx \times t$$

$$1 - \sin 1 = (1 + \sin 1)(d)(1)$$

$$[1 - \sin 1] / [1 + \sin 1] = 1585 / 1.866 = 117.7 = M = d$$

$$117.7 / 12 = 9.8083 = g \text{ (Figure 5).}$$

Static Equilibrium

Joints 1,3,7

$$2^n - 1$$

$$[(dM/dt)^n - M] / c = j_{\text{critical}} / c = Ec$$

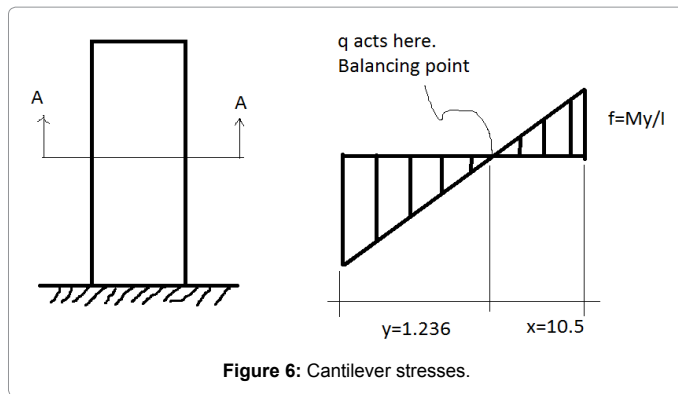


Figure 6: Cantilever stresses.

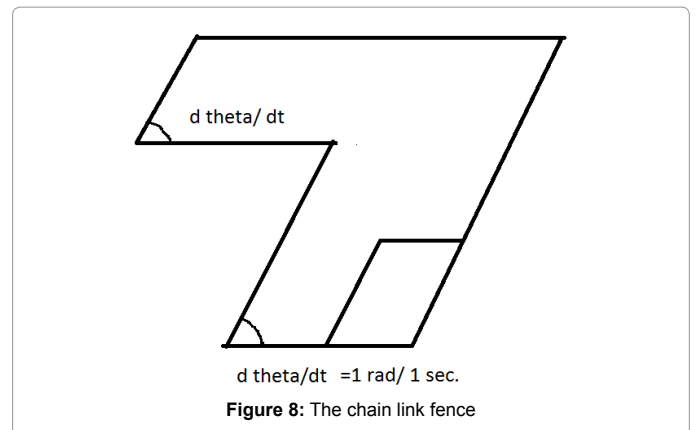


Figure 8: The chain link fence

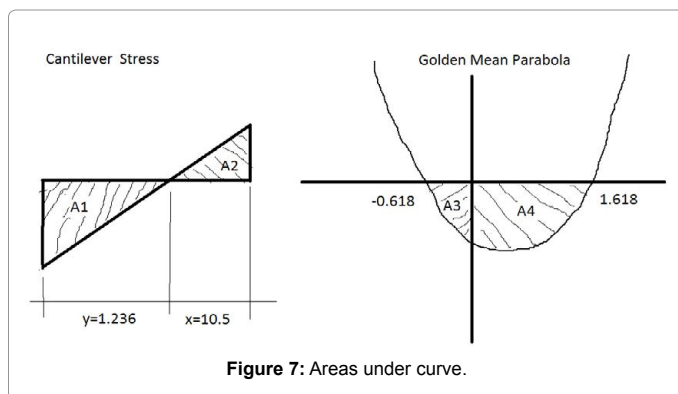


Figure 7: Areas under curve.

10.5/0.0809=11.72~1118 Last element in the periodic table of the elements (Figure 7).

$$A1 = 1/2 bh = 1/2 (1/81)(1.236) = 7.639$$

$$A2 = 1/2(129.5)(10.5) = 679.22$$

$$\Sigma[A1+A2] = 6.7922 - 7.639 = 0.8468$$

$$A3 = \int_{(-0.618-0)} t^2 - t - 1 = 2t^3/3 - 3/2t - t = E^2/2$$

$$E = 0$$

$$A4 = \int(0-1.618) = 1.618$$

$$\Sigma[A3+A4] = 1.618$$

$$\Sigma[A1+A2] / \{A3+A4\} =$$

$$0.8468/1.618 = 2.998 \sim c \text{ (Figure 8).}$$

$$d^2y/dt^2 = M/EI$$

$$d\theta/dt = M/EI$$

$$1 \text{ rad} = M/(0.4233)(1/Y)$$

$$Y = 202$$

$$M = 1/1201$$

$$1/12.01 \times 360 = 2.9975 = c$$

$$\delta = \int M/EI \times dx$$

$$= \Sigma[A1+A2]$$

$$\delta/t = \delta/1.618 = \Sigma[A1+A2] / \Sigma[A3+A4] = d/t = v = c$$

Divine Mercy Icon

$$E = Mc^2$$

$$MaE/c^2$$

$$dM/dt \text{ ad } E/dt \times 1/[2c]$$

$$2\alpha 1(1/2(3))$$

$$12 \alpha 1$$

$$12 \text{ Apostles}$$

$$1/12 = t = \text{Judas}$$

And

$$F = t = d$$

$$Fd = 1 + F = 1 + \sin 60^\circ$$

$$Fd - F - 1 = 0$$

$$(\sin \theta)(\sin \theta) - F - t = 0$$

$$1/s = Fd$$

$$1 = Fds$$

$$t = E = Fds$$

$$t = E = Fs^2$$

$$= F(4/3)^2$$

$$= F(16/9) = F \sqrt{\pi}$$

$$s^2 = \sqrt{\pi}$$

$$s^4 = \pi = I = \text{moment of inertia} = t$$

$$s^4 = \Omega = \text{linear operator}$$

Now, for the Great Abomination of Revelation:

$$y = \delta$$

$$dy/dt = \theta$$

$$d^2y^2/dt^2 = d\theta/dt = M/EI$$

$$d^3y/dt^3 = dM/EI \text{ dt} = V/EI$$

$$= 2/[0.4233](4/3) = 345.359$$

April 3, 2005 @ 3:36 P.M. Date and Time Idol of the oppressor was installed at the Cathedral of the Immaculate Conception in Saint John: Divine Mercy Sunday.

$EVI = \text{Energy (Shear)} \times \text{Inertia} \times \text{Length} = (1)(2)(1/202)(1/2) = 1/202 = 1/Y = 1/E = t$ Time is evil (Figure 9).

Conjugate Beam Theory

The slope ($t = 1.0123$ rads) and the deflection ($E = 0$) is the shear in the real beam is the shear (58) and the moment (0) in the conjugate beam [1,2].

In the Conjugate Beam, the,

Shear $V = \text{Slope } \theta$

and

Deflection $\delta = \text{Moment } M$

so,

$V = dM/dt = 2 = \sqrt{3}/\sin \theta$

$\theta = dE/dt = 1$

$\delta = t = \int M/EI \times dx = 1 - \sin 1 = Fd = 1682$

$M = 1 - \sin 1 = Fd = 0.1585$

$EaMc^2$

Derivative:

$dE/dt = dM/dt \times 2c$

$\theta = V \times 2 \times \delta$

$\theta = 2\sqrt{3}/\sin \theta \times \delta$

$\theta/\delta = 4.00 = t = 40\%$ of a cycle

$\delta = \theta/4$

$\theta = 1/2\pi$

$\delta = 0.1592/4 = 0.0398 = 1/251 = 1/T = t$

$\delta = t = \gamma = E$

$= -[1.333 - 1]$

$= -0.333$

$= -1/c$

$s - t = -1/c$

$[|E|]t[\sin \theta] - t = -1/c$

$t[E \sin \theta - 1] = -1/c$

$1[\sin \theta - 1] = -0.333$

$\sin \theta = 0.777$

$\theta = 2.98 \sim c$

$\sin \theta - 1 = -1/c$

$1 - \sin \theta = 1/c$

$\theta = 0.7296$ rads

$41.8^\circ/360^\circ = 116.1 = M$

$1/c = E \times t$

$1/(tc) = E$

$1/c \div (1/E) = E$

$E/c = t$

$E/c = 1$

$E = 3$ (Figures 10 and 11).

Axial load

$\delta = PL/AE$

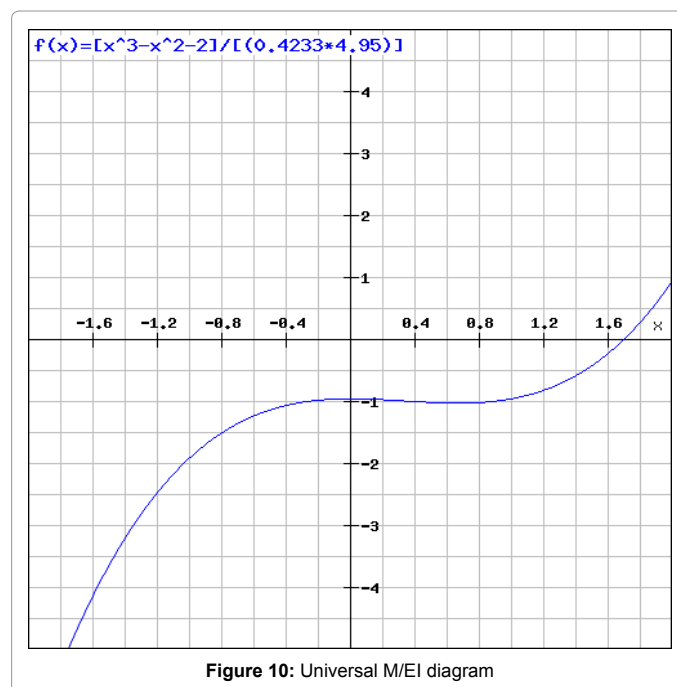
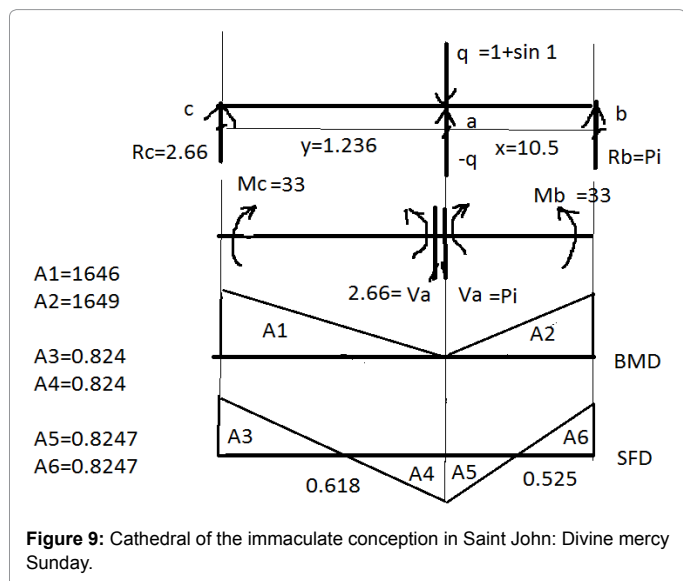
$2t - 1 = w = 0$

$2t = 1$

$t = 1/2$

The Universal Beam

$\int BMD = \int_{(-1 \text{ to } 2)} t^3 - t^2 - t - 2$



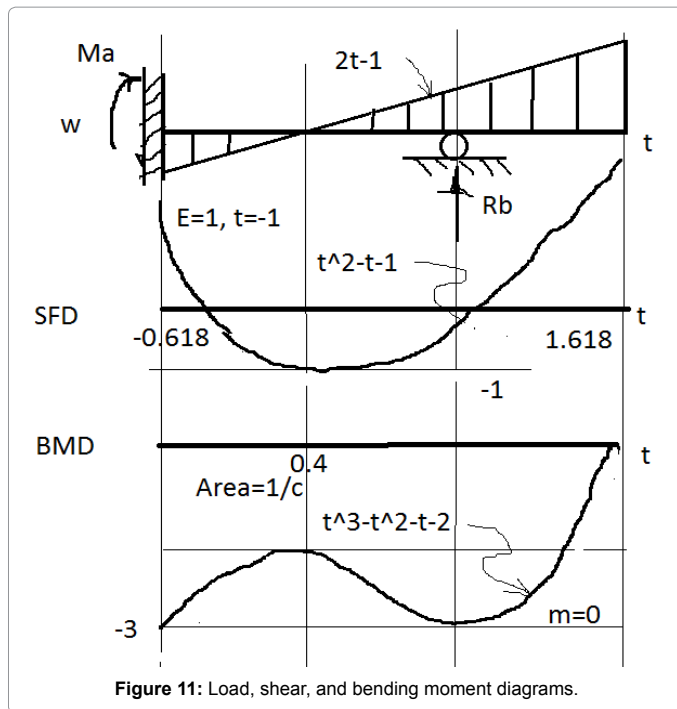


Figure 11: Load, shear, and bending moment diagrams.

Moment

$$t^3 - t^2 - t - 2 = 0$$

$$(1/2)^3 - (1/2)^2 - (1/2) - 2 = 26.25$$

$$\delta = 1 = P(117.27) / [A (0.4233)]$$

$$P/A = \sigma = 3.587$$

$$\sigma / M_p = 3587 / 938 = 26.15$$

Work

$$W_{\text{Internal}} = \int_{(0-L)} M^2 / 2EI \, dx$$

(where M = moment in this particular case.)

$$W = (F x)^2 / 2EI \, dx$$

$$= F^2 (2/3 x^3) / (2EI) \text{ from } 0 - L$$

F is the Superforce = 2.667

$$= 2.667^2 (2/3)(118) / 2 (0.4233)(1/202)$$

$$= 1/127.5 = 1/\rho.$$

And the Ether:

Shear Modulus G

$$1/G = 2(1+\nu)/E$$

$$= 2(1.27)/0.4233$$

$$= 6 = 1/G$$

$$G = 0.167 = \text{monoatomic gas } \gamma$$

$$W_i = K \int_{(0-L)} V^2 / (2GA) \, dx$$

$$1/\rho = K/A \, dx / (2(1/6))$$

$$K/A \, dx = 941$$

K for a square = 1.2

$$1.2/A \, dx = 941$$

$$A = x^2$$

$$1.2 x^{-2} dx = 941 \, dx$$

$$1.2 (-1/x) = 941.$$

$$x = 127.5 = \rho$$

$$A = x^2 = 127.5^2 = 1625 \text{ (~1623 Navier-Stoke's Solution)}$$

$$\Sigma W_{\text{Internal}} = W_{\text{Axial}} + W_{\text{Bending}} + W_{\text{Shear}} + W_{\text{Twist}}$$

$$\Sigma W_{\text{Internal}} = 4 (1/\rho) = 3.13 \sim \pi = t.$$

Real work

$$\Sigma W_{\text{Internal}} = \text{Mom.} + V$$

$$= P/2 (\text{Average Mass} + 1)$$

(Note: Atomic Number of Element Te = 58)

$$= 2.667/2 [58 + 1]$$

$$= 1.334(59) = 78.667 = 1/\rho$$

For a Virtual Displacement, $\delta = 1$, the Total work done = 0

$$\Sigma W_{\text{External}} = \Sigma W_{\text{Internal}}$$

$$\Sigma W_{\text{Total}} = 4 \Sigma W_{\text{Internal}}$$

$$F \times d = (2.667)(118) = 4/\rho = 4/127.6 = \pi$$

d = 118 = Mass of the Periodic Table of the Elements.

Conclusion

So we see that Statics and mechanics as applied to a force polygon provide useful insights into physical constants. In addition, we show that the physical and spiritual are inseparable. Energy methods are useful in understanding the Ether as a monoatomic gas.

References

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2. Laursen HI (1969) Structural analysis McGraw-Hill, The City University of New York, New York.

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